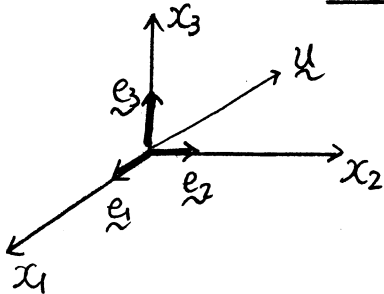


Given a continuum medium subjected to external loading, we want to find:

1. the displacement field $u_i(\underline{x})$ — vector
2. the strains field $\epsilon_{ij}(\underline{x})$ } tensor
3. the stress field $\sigma_{ij}(\underline{x})$ } (rank 2)

§1. What is a vector?



We can represent a vector by

- an arrow (in a figure)
- a symbol \sim
- its components (coordinates)

$$\underline{u} = u_1 \underline{e}_1 + u_2 \underline{e}_2 + u_3 \underline{e}_3 = \sum_{i=1}^3 u_i \underline{e}_i$$

index notation: repeated index is summed over from 1 to 3 (dummy index)

$$\underline{u} = u_i \underline{e}_i$$

Notice here we specify a vector \underline{u} by linear combination of three (unit) vectors \underline{e}_i . $\underline{e}_1, \underline{e}_2, \underline{e}_3$ form a coordinate system.

If we choose a different set of (unit) vectors $(\underline{e}'_1, \underline{e}'_2, \underline{e}'_3)$ as our coordinate system, then the same vector \underline{u} will have different coordinates (u'_1, u'_2, u'_3)

$$\underline{u} = u_i \underline{e}_i = u'_i \underline{e}'_i$$

define $Q_{ij} = (\underline{e}_i \cdot \underline{e}_j)$ (dot product)

* Q_{ij} forms an orthogonal matrix

$$Q_{ij} Q_{kj} = \delta_{ik}$$

notice j is a dummy variable,

$$\delta_{ik} = \begin{cases} 0 & i \neq k \\ 1 & i = k \end{cases}$$

is called Kronecker delta.

In Matrix notation

$$Q \cdot Q^T = I$$

↑
transpose

← Identity matrix

notice $(\underline{e}_i \cdot \underline{e}_j) = \delta_{ij} = \begin{cases} 0 & i \neq j \\ 1 & i = j \end{cases}$

$$(\underline{e}_i \cdot \underline{e}_j') = \delta_{ij}$$

$$u_i \underline{e}_i' = u_i \underline{e}_i$$

multiply both sides by \underline{e}_j' (dot product)

$$u_j' = u_i (\underline{e}_j' \cdot \underline{e}_i)$$

$$u_j' = Q_{ji} u_i$$

(i is dummy variable, summed over 1, 2, 3
 j is a free index, can be 1, 2, 3)

↕

$$u_1' = Q_{11} u_1 + Q_{12} u_2 + Q_{13} u_3$$

$$u_2' = Q_{21} u_1 + Q_{22} u_2 + Q_{23} u_3$$

$$u_3' = Q_{31} u_1 + Q_{32} u_2 + Q_{33} u_3$$

↕

$$u_i' = Q_{ij} u_j$$

(i is a free index,
 j is a dummy index)

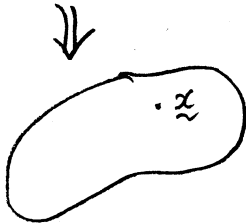
The same equation(s) can be expressed using different choices of indices.

§2. displacement (vector) field $u_i(x)$



a material point in the undeformed state is specified by a vector

$$\underline{X} = X_i \underline{e}_i$$



The same material point in the deformed state is specified by another vector

$$\underline{x} = x_i \underline{e}_i$$

displacement vector $\underline{u} \equiv \underline{x} - \underline{X}$

$\underline{u}(\underline{x})$ is a vector defined for every material point \underline{x} , and is called a vector field.

i.e. a vector as a function of another vector.

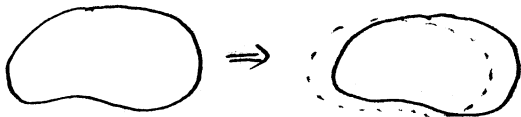
Infinitesimal elasticity, we assume $|\underline{u}| \ll 1$, so that we do not distinguish $\underline{u}(\underline{x})$ v.s. $\underline{u}(\underline{X})$

This is a great simplification.

No longer valid for large deformation, which is treated in continuum mechanics.

§3. strain field $\epsilon_{ij}(\underline{x})$

a displacement field $u_i(\underline{x})$ does not necessarily lead to "deformation."

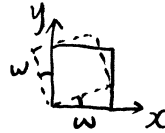


rigid-body translation

$$\underline{u}(\underline{x}) = \underline{u}_0 \text{ constant vector}$$



rigid-body rotation



$$u_y = +\omega \cdot x$$

$$u_x = -\omega \cdot y$$

(to first order of x, y)

Taylor expand displacement field $u_i(\underline{x})$
up to 1st order.

$$u_i(\underline{x}) = u_i^0 + \frac{\partial u_i}{\partial x_j} dx_j$$

index notation

$$u_{i,j} \equiv \frac{\partial u_i}{\partial x_j}$$

$$= u_i^0 + u_{i,j} dx_j$$

$$= u_i^0 + \frac{1}{2} (u_{ij} + u_{j,i}) dx_j + \frac{1}{2} (u_{ij} - u_{j,i}) dx_j$$

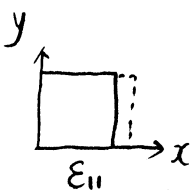
$$u_i = u_i^0 + \epsilon_{ij} dx_j + \omega_{ij} dx_j$$

$$\epsilon_{ij} \equiv \frac{1}{2} (u_{ij} + u_{j,i})$$

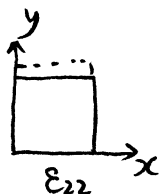
strain: $\epsilon_{ij} = \epsilon_{ji}$ symmetric

$$\omega_{ij} \equiv \frac{1}{2} (u_{ij} - u_{j,i})$$

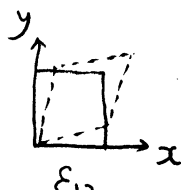
rotation: $\omega_{ij} = -\omega_{ji}$ anti-symmetric



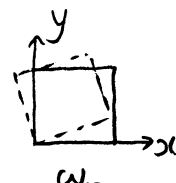
strain



strain



strain



rotation

$$\epsilon_{ij} \equiv \frac{1}{2} (u_{i,j} + u_{j,i}) = \frac{1}{2} \left(\frac{\partial u_i}{\partial x_j} + \frac{\partial u_j}{\partial x_i} \right)$$

This is the strain component when we express vectors \underline{u} and \underline{x} in the coordinate system of $\underline{e}_1, \underline{e}_2, \underline{e}_3$

Q: What happens if we choose a different coordinate system?

$$\underline{u} = u'_i \underline{e}'_i$$

$$\underline{x} = x'_i \underline{e}'_i$$

$$\epsilon'_{ij} \equiv \frac{1}{2} \left(\frac{\partial u'_i}{\partial x'_j} + \frac{\partial u'_j}{\partial x'_i} \right)$$

Q: how does ϵ_{ij} transform with a change of coordinate system?

We can show that

$$\epsilon'_{ij} = Q_{ip} Q_{jq} \epsilon_{pq}$$

p, q are dummy indices
 i, j are free indices.

* recall $u'_i = Q_{ij} u_j$

The "rule" is that: the first index of Q is the free index
the second index of Q is the dummy.

* The relation in the box can be shown through the "chain rule"

notice that $x'_j = Q_{jq} x_q$

(be careful!) $\rightarrow x_q = Q_{jq} x'_j$

— because \underline{x} is a vector

— because $Q \cdot Q^T = I$

Q 's inverse is its transpose

we have $\frac{\partial x_q}{\partial x'_j} = Q_{jq}$

$$\frac{\partial}{\partial x'_j} f = \left(\frac{\partial}{\partial x_q} f \right) \cdot \left(\frac{\partial x_q}{\partial x'_j} \right) = Q_{jq} \frac{\partial}{\partial x_q} f$$

$f_{,j} = Q_{jq} f_{,q}$ — spatial derivatives transform as vector

E_{ij} is a matrix of 9 numbers that transform according to $E'_{ij} = Q_{ip} Q_{jq} E_{pq}$ — which is the definition of a rank-2 tensor.

* An arbitrary ranked tensor $A_{ijklm\dots n}$ is a set of numbers that transform as

$$A'_{ijklm\dots n} = Q_{ip} Q_{jq} Q_{kr} Q_{ls} Q_{mt} \dots Q_{nu} A_{pqrst\dots u}$$

Always: the first index of Q is free
the second index of Q is dummy

$$Q_{ip} \equiv (\underline{e}'_i \cdot \underline{e}_p)$$

* The differential operator is a vector

$$\underline{\nabla} = \frac{\partial}{\partial x_i} (\cdot) \underline{e}_i \quad \left(\text{e.g. } \underline{\nabla} f = \left(\frac{\partial f}{\partial x_i} \right) \underline{e}_i \right)$$

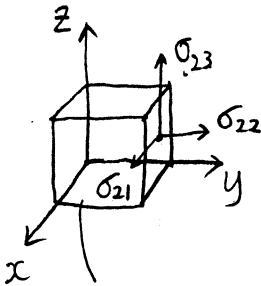
* A tensor can also be expressed in vector notation

$$\begin{aligned} \underline{\underline{\varepsilon}} &= \varepsilon_{ij} (\underline{e}_i \otimes \underline{e}_j) && \underline{e}_i \otimes \underline{e}_j \text{ is a tensor product} \\ &= \frac{1}{2} \left[(\underline{\nabla} \underline{u}) + (\underline{\nabla} \underline{u})^T \right] && \text{- also called dyadic notation} \end{aligned}$$

Reading assignment:

Sadd Section 1.1 - 1.8

* see section 1.9 for complications that can arise in curvilinear (e.g. cylindrical or polar) coordinate systems.

§4. stress field.

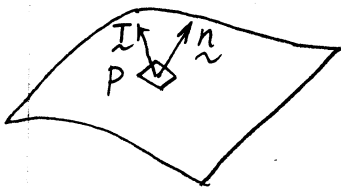
"the stress cube"

σ_{ij} = force per unit area on i -th face
in j -th direction (*note correction
from previous version!)

* This is the Cauchy stress in
continuum mechanics.

Given the stress field, we can obtain the traction force \underline{T}
per unit area on any surface element with normal
vector \underline{n} =

$$\underline{T}_j = \sigma_{ij} n_i$$



We can show (but not here) that the stress is also
a rank-2 (symmetric) tensor, i.e. it transforms as

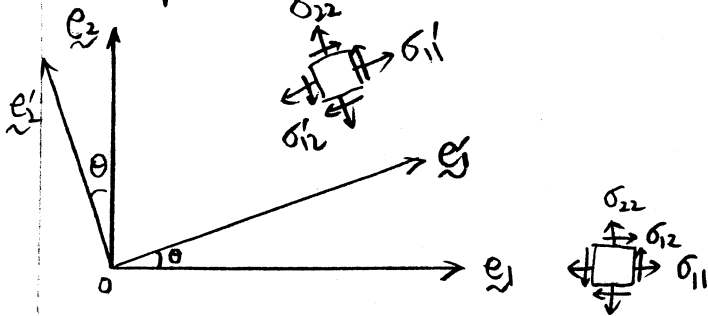
$$\sigma'_{ij} = Q_{ip} Q_{jq} \sigma_{pq}$$

$$\sigma_{ij} = \sigma_{ji}, \quad \sigma'_{ij} = \sigma'_{ji}$$

* both \underline{T} and \underline{n} are vectors. i.e. $n'_i = Q_{ip} n_p$
 $T'_j = Q_{j2} T_2$

In fact, we can use these two relations to prove the
relation in the box.

Example 1: Rotation around z-axis



$$Q_{11} = \underline{e}'_1 \cdot \underline{e}_1 = \cos\theta \quad Q_{12} = \underline{e}'_1 \cdot \underline{e}_2 = \sin\theta$$

$$Q_{21} = \underline{e}'_2 \cdot \underline{e}_1 = -\sin\theta \quad Q_{22} = \underline{e}'_2 \cdot \underline{e}_2 = \cos\theta$$

$$Q_{13} = Q_{23} = Q_{31} = Q_{32} = 0$$

$$Q_{33} = 1$$

$$Q = \begin{bmatrix} \cos\theta & \sin\theta & 0 \\ -\sin\theta & \cos\theta & 0 \\ 0 & 0 & 1 \end{bmatrix}$$

$$\sigma'_{11} = \cos^2\theta \sigma_{11} + \sin^2\theta \sigma_{22} + 2\sin\theta\cos\theta \sigma_{12}$$

$$\sigma'_{22} = \sin^2\theta \sigma_{11} + \cos^2\theta \sigma_{22} - 2\sin\theta\cos\theta \sigma_{12}$$

$$\sigma'_{12} = -\sin\theta\cos\theta \sigma_{11} + \sin\theta\cos\theta \sigma_{22} + (\cos^2\theta - \sin^2\theta) \sigma_{12}$$

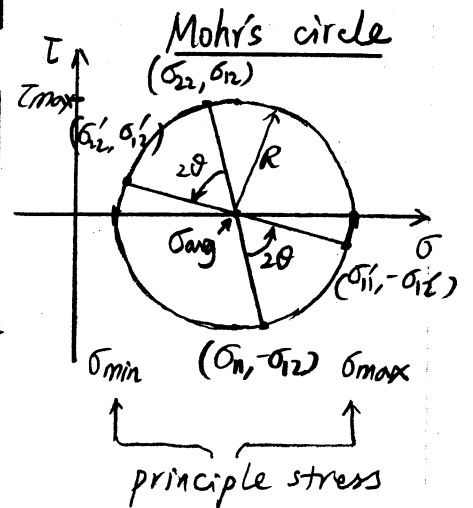
$$\leftarrow \sigma'_{ij} = Q_{ip} Q_{jq} \sigma_{pq}$$

or equivalently by

$$\sigma'_{11} = \frac{\sigma_{11} + \sigma_{22}}{2} + \frac{\sigma_{11} - \sigma_{22}}{2} \cos 2\theta + \sigma_{12} \sin 2\theta$$

$$\sigma'_{22} = \frac{\sigma_{11} + \sigma_{22}}{2} - \frac{\sigma_{11} - \sigma_{22}}{2} \cos 2\theta - \sigma_{12} \sin 2\theta$$

$$\sigma'_{12} = -\frac{\sigma_{11} - \sigma_{22}}{2} \sin 2\theta + \sigma_{12} \cos 2\theta$$



$$\sigma_{avg} = \frac{\sigma_{11} + \sigma_{22}}{2}$$

$$\tau_{max} = R = \sqrt{\left(\frac{\sigma_{11} - \sigma_{22}}{2}\right)^2 + \sigma_{12}^2}$$

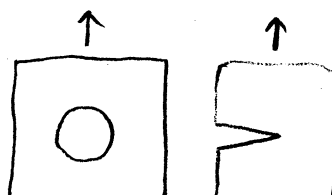
$$\sigma_{max} = \sigma_{avg} + R$$

$$\sigma_{min} = \sigma_{avg} - R$$

Stress points rotate twice as fast in Mohr's circle as in real space (2θ v.s. θ)

* Strain tensor satisfies a similar transformation rule. Hence there exist a Mohr's circle for strain.

* These relations are useful in polar coordinates, which are convenient



for these problems ←